

A Local Mesh Modification Strategy for Interface Problems with Application to Shape and Topology Optimization

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A Local Mesh Modification Strategy for Interface Problems with Application to Shape and Topology Optimization

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Abstract

We present and analyze a new finite element method for solving interface problems on a triangular grid. The method locally modifies a given triangulation such that the interfaces are accurately resolved and the maximal angle condition holds. Therefore, optimal order of convergence can be shown. Moreover, an appropriate scaling of the basis functions yields an optimal condition number of the stiffness matrix. The method is applied to an optimal design problem for an electric motor where the interface between different materials is evolving in the course of the optimization procedure.

1 Motivation

Our research is motivated by the design optimization of an electric motor by means of topology and shape optimization. We are interested in finding the optimal distribution of two materials (usually ferromagnetic material and air) within a fixed design subdomain of an electric motor, see, e.g. [1]. We employ a two-dimensional model for the electric motor, which is widely used for this kind of applications. In the optimization procedure, one usually starts with an initial guess and then uses shape sensitivities or topological sensitivities to gradually improve the initial design. In the course of this optimization procedure, the interface between the two subdomains evolves. For computing the sensitivities that steer the optimization process, it is necessary to solve the state equation and the adjoint equation in each optimization iteration, which is usually done by the finite element method. Besides remeshing in every iteration, which is very costly, and advecting the whole mesh in every step of the optimization procedure, which may cause self-intersection of the mesh, there exist several

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other methods in the literature which can deal with these kinds of interface problems. We mention the XFEM, which uses local enrichment of the finite element basis, and the unfitted Nitsche method.

In [2], the authors introduce a locally modified parametric finite element method based on a quadrilateral mesh with a patch structure. We present an adaptation of this method to the case of finite elements on triangular meshes. One advantage of this kind of method over the ones mentioned before is that this method has a fixed number of unknowns independently of the position of the interface relative to the mesh. The given mesh is modified only locally near the material interface. The method is relatively easy to implement and we can show optimal order of convergence.

2 A local mesh modification strategy for Interface Problems

We introduce the method for the potential equation in a bounded, polygonal computational domain $\Omega \subset \mathbb{R}^2$ consisting of two non-overlapping subdomains, $\bar{\Omega} = \bar{\Omega}_1 \cup \bar{\Omega}_2$, $\Omega_1 \cap \Omega_2 = \emptyset$, which represent two materials with different material coefficients $\kappa_1, \kappa_2 > 0$. On the material interface $\Gamma := \bar{\Omega}_1 \cap \bar{\Omega}_2$, we have to require that the solution as well as the flux are continuous. For simplicity, we assume homogeneous Dirichlet boundary conditions on $\partial\Omega$. The problem reads as follows:

$$\begin{aligned} -\operatorname{div}(\kappa_i \nabla u) &= f && \text{in } \Omega_i, \quad i = 1, 2, \\ [u] &= 0 && \text{on } \Gamma, \\ \left[\kappa \frac{\partial u}{\partial n} \right] &= 0 && \text{on } \Gamma, \\ u &= 0 && \text{on } \partial\Omega, \end{aligned} \tag{1}$$

where we assume that the boundaries of the two subdomains as well as the right hand side f are sufficiently regular such that $u \in H_0^1(\Omega) \cap H^2(\Omega_1 \cup \Omega_2)$, that means that the restrictions of $u \in H_0^1(\Omega)$ to Ω_1 and Ω_2 belong to $H^2(\Omega_1)$ and $H^2(\Omega_2)$, respectively, see, e.g., [3]. It is well-known that, when using standard finite element methods, the interface must be resolved by the mesh in order to obtain optimal convergence rates of the approximate solution u_h to the true solution u in the L^2 and H^1 norms as the mesh parameter h tends to zero, see also [2]. The discretization error estimate is usually shown using an interpolation error estimate. A condition that is sufficient and necessary for such an interpolation error estimate is that all interior angles of triangles of the mesh are bounded away from 180° (maximum angle condition), see [4].

2.1 Preliminaries

Let \mathcal{T}_h be a shape-regular and quasi-uniform subdivision of Ω into triangular elements, and let us denote the space of globally continuous, piecewise linear functions on \mathcal{T}_h by V_h . We assume that \mathcal{T}_h has been obtained by one uniform refinement of a coarser mesh \mathcal{T}_{2h} . By this assumption, \mathcal{T}_h has a patch-hierarchy, i.e., always four elements $T_1, T_2, T_3, T_4 \in \mathcal{T}_h$ can be combined to one larger triangle $T \in \mathcal{T}_{2h}$. We will refer to this larger element as the makro element or

patch. We assume further that the mesh of makro elements \mathcal{T}_{2h} is such that, for each makro element T , the interface Γ either does not intersect the interior of T , or such that Γ intersects T in exactly two distinct edges or that it intersects T in one vertex and in the opposite edge. For a smooth enough interface Γ , this assumption can always be enforced by choosing a fine enough makro mesh \mathcal{T}_{2h} . We consider a makro element $T \in \mathcal{T}_{2h}$ to be cut by the interface if the intersection of the interior of makro element with interface is not the empty set.

2.2 Description of the method

The method presented in this paper is a local mesh adaptation strategy, meaning that only makro elements close to the interface Γ will be modified. Given the hierarchic structure of the mesh, on every makro element we have four elements of the mesh \mathcal{T}_h and six vertices, see Fig. 1(a),(b). The idea of the method is the following: For each makro element that is cut by the interface, move the points P_4 , P_5 and P_6 along the corresponding edges in such a way that, on the one hand, the interface is resolved accurately, and, on the other hand, all interior angles in the four triangles are bounded away from 180° . For a makro element T that is cut by the interface, we distinguish four different configurations as follows:

In the case where the makro element is cut by the interface in two distinct edges, we denote the vertex of the makro element where these two edges meet by P_1 , and the other two vertices in counter-clockwise order by P_2 and P_3 . The parameters $s, t, r \in [0, 1]$ represent the positions of the points P_4, P_5, P_6 along the corresponding edges by

$$P_4(s) = P_1 + s \frac{P_2 - P_1}{|P_2 - P_1|}, \quad P_5(t) = P_2 + t \frac{P_3 - P_2}{|P_3 - P_2|}, \quad P_6(r) = P_1 + r \frac{P_3 - P_1}{|P_3 - P_1|}.$$

The parameters r and s will always be chosen in such a way that the intersection points of the interface and the edges P_1P_3 and P_1P_2 are the points P_6 and P_4 , respectively. Thus, we identify the position of the interface relative to the makro element T by the two parameters r, s . We choose the parameter t such that a maximal angle condition is satisfied as follows:

Configuration A: $0 < r, s \leq 1/2$. Set $t = 1/2$.

Configuration B: $1/2 < r, s < 1$. Set $t = 1 - s$.

Configuration C: $0 < s \leq 1/2 < r < 1$ or $0 < r \leq 1/2 < s < 1$. Set $t = 1/2$.

The case where the makro element is cut in one vertex and the opposite edge has to be considered separately. We denote the vertex of the makro element where it is cut by the interface by P_2 and the other vertices, in counter-clockwise ordering, by P_3 and P_1 , see Fig. 1(b). The location of the interface is given by the position of the point P_6 on the edge between P_3 and P_1 . In this case, we also need to rearrange the triangles T_2 and T_4 .

Configuration D:

Configuration D1: $0 < r \leq 1/2$. Set $s = r$ and $t = 1/2$.

Configuration D2: $1/2 < r < 1$. Set $s = 1/2$ and $t = r$.

With this setting, it is possible to show the required maximal angle condition on the reference patch \hat{T} defined by the outer makro vertices $\hat{P}_1 = (0, 0)^T$, $\hat{P}_2 = (1, 0)^T$, $\hat{P}_3 = (1/2, \sqrt{3}/2)^T$.

Lemma 1. *All angles in triangles of the reference patch \hat{T} are bounded by 150° independent of the parameters $r, s \in [0, 1]$.*

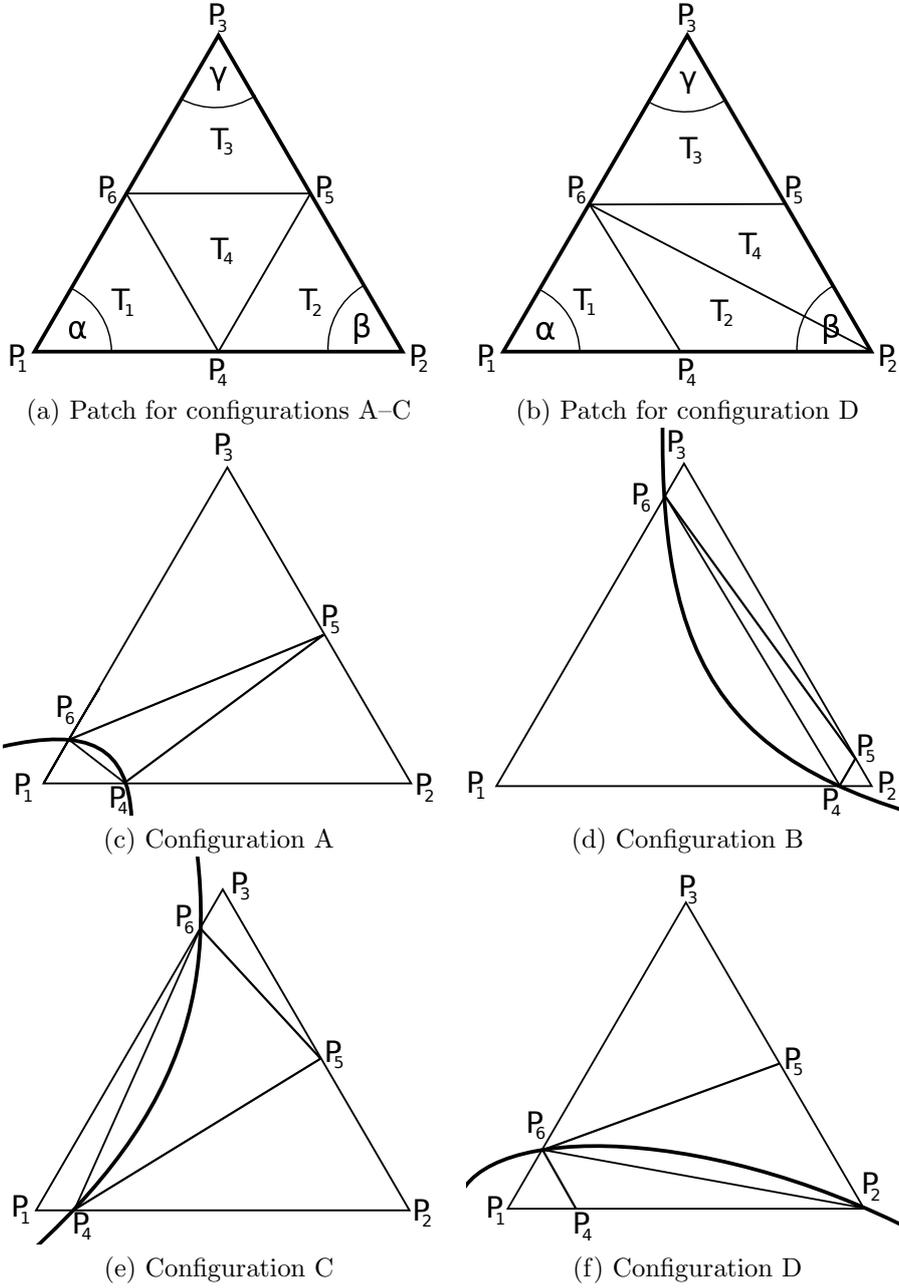


Figure 1: (a), (b): Patches for different configurations. (c)-(f): Different configurations of mesh points depending on position of the interface.

Proof. We have to ensure for each of the four subtriangles $\hat{T}_1, \hat{T}_2, \hat{T}_3, \hat{T}_4$ that all of their three interior angles are not larger than 150° . In Configuration A–C, the sub-triangles \hat{T}_1, \hat{T}_2 and \hat{T}_3 all have one angle of 60° . Obviously, the remaining two angles are bounded from above by 120° . The same holds true for the sub-triangles \hat{T}_1 and \hat{T}_3 in Configuration D.

For three points A, B, C in \mathbb{R}^2 , define

$$\angle(A, B, C) := \cos^{-1} \left(\frac{(A - B, C - B)}{|A - B| |C - B|} \right)$$

the interior angle of the triangle with vertices A, B, C at point B .

Configuration A: For $r, s \in (0, 1/2]$, we get for the angle in point P_4 that

$$\angle(P_6, P_4, P_5) < \angle(P_1, P_4, P_5) = 180^\circ - \angle(P_5, P_4, P_2) \leq 180^\circ - \angle(P_5, P_1, P_2).$$

Since the reference patch \hat{T} is equilateral, it holds $\angle(P_5, P_1, P_2) = \alpha/2$. Analogously, we get for the angle in point P_6 that $\angle(P_5, P_6, P_4) < 180^\circ - \alpha/2$. It is easy to see that the angle in point P_5 increases with r, s and thus is maximized for $r = s = 1/2$, which yields that $\angle(P_4, P_5, P_6) \leq \angle(P_4(1/2), P_5, P_6(1/2)) = 180 - \beta - \gamma = \alpha$. Here we used that, for $r = s = t = 1/2$, the four sub-triangles are congruent.

Configuration B: Note that, by the special choice of s, t , in this case we have that the line going through P_4 and P_5 is parallel to the edge connecting P_1 and P_3 for all values of $s \in (1/2, 1)$. Thus, we have

$$\begin{aligned} \angle(P_4, P_5, P_6) &\leq \angle(P_4, P_5, P_3) = 180^\circ - \gamma \quad \text{and} \\ \angle(P_4, P_5, P_6) &= 180^\circ - \gamma - \angle(P_6, P_5, P_3) \\ &\geq 180^\circ - \gamma - \angle(P_6(1/2), P_5(1/2), P_3) = 180^\circ - \gamma - \beta = \alpha. \end{aligned}$$

The angles in P_4 and in P_6 must also be bounded from above by $180^\circ - \alpha = 120^\circ$.

Configuration C: We consider the case where $r \in (1/2, 1)$ and $s \in (0, 1/2]$. The reverse case is treated analogously. For the angle in the fixed point $P_5 = P_5(1/2) = (P_2 + P_3)/2$, we get the estimates

$$\begin{aligned} \angle(P_4, P_5, P_6) &\leq \angle(P_4, P_5, P_3) \leq \angle((P_4(1/2), P_5, P_3) = 180^\circ - \gamma, \\ \angle(P_4, P_5, P_6) &\geq \angle(P_4, P_5, P_6(1/2)) \geq \angle(P_1, P_5, P_6(1/2)) = \angle(P_5, P_1, P_2) = \alpha/2. \end{aligned}$$

Thus, the angles $\angle(P_6, P_4, P_5)$ and $\angle(P_5, P_6, P_4)$ are also bounded from above by $180^\circ - \beta/2$.

Configuration D: We consider only Configuration D1, the corresponding result for Configuration D2 follows analogously. Due to the choice of the parameter s , the line going through P_4 and P_6 is parallel to the edge connecting P_2 and P_3 for all values of r . We need to consider triangles T_2 and T_4 . In T_2 , $\angle(P_6, P_4, P_2) = 180^\circ - \beta$ and, therefore, the other two angles are bounded by β . In T_4 , we have for $r \in (0, 1/2]$ that

$$\begin{aligned} \angle(P_6, P_2, P_5) &\leq \beta, \\ \angle(P_2, P_5, P_6) &\leq \angle(P_2, P_5, (P_3 + P_1)/2) = 180 - \beta, \\ \angle(P_5, P_6, P_2) &\leq \angle(P_3, P_6, P_4) = 180 - \gamma. \end{aligned}$$

Finally, noting that $\alpha = \beta = \gamma = 60^\circ$ yields the statement of the lemma. \square

Remark 1. *Due to the assumption that the makro mesh is shape-regular, we obtain a maximal angle condition (with a different bound) for all triangles of the mesh \mathcal{T}_h .*

Now we are in the position to show an a priori error estimate for the finite element solution u_h . Since we have the maximum angle condition of Lemma 1, we get the interpolation error estimates

$$\|\nabla^k(v - I_h v)\|_{L^2(T)} \leq c h_{T,max}^{2-k} \|\nabla^2 v\|_T, \quad k = 0, 1, \quad (2)$$

where $I_h : H^2(T) \rightarrow V_h|_{\overline{T}}$ denotes the Lagrangian interpolation operator, c is a positive generic constant, and $h_{T,max}$ is the maximum edge length of the triangle $T \in \mathcal{T}_h$, see, e.g., [5]. In the case where the interface Γ is not polygonal but smooth with C^2 parametrization, and an element of the mesh \mathcal{T}_h is intersected by Γ , the solution u is not smooth across the interface and, hence, estimate (2) cannot be applied. However, the same estimate with $k = 1$ was shown in [6]. These interpolation error estimates allow to show the following a priori error estimate [6].

Theorem 1. *Let $\Omega \subset \mathbb{R}^2$ be a domain with convex polygonal boundary, split into $\Omega = \Omega_1 \cup \Gamma \cup \Omega_2$, where Γ is a smooth interface with C^2 -parametrization. We assume that Γ divides Ω in such a way that the solution u belongs to $H_0^1(\Omega) \cap H^2(\Omega_1 \cup \Omega_2)$ and satisfies the stability estimate $\|u\|_{H^2(\Omega_1 \cup \Omega_2)} \leq c_s \|f\|$. Then, for the corresponding modified finite element solution $u_h \in V_h$, we have the estimates*

$$\|\nabla(u - u_h)\|_{L^2(\Omega)} \leq C h \|f\| \quad \text{and} \quad \|u - u_h\|_{L^2(\Omega)} \leq C h^2 \|f\|.$$

3 Condition number

The procedure of Section 2 guarantees that no angle of the modified mesh becomes too large. However, it may happen that some angles in the triangulation are getting arbitrarily close to zero, which usually yields a bad condition of the finite element system matrix. This problem was also addressed in [2] for the case of quadrilateral elements, and we can adapt the procedure to the triangular case.

The idea consists in a hierarchical splitting of the finite element space $V_h = V_{2h} + V_b$ into the standard piecewise linear finite element space on the makro mesh \mathcal{T}_{2h} and the space of ‘‘bubble’’ functions in V_b which vanish on the nodes of the makro elements. Let $\{\phi_h^1, \dots, \phi_h^{N_h}\}$ be the nodal basis of the space V_h . Any function $v_h \in V_h$ can be decomposed into the sum of a function $v_{2h} \in V_{2h} = \text{span}\{\phi_{2h}^1, \dots, \phi_{2h}^{N_{2h}}\}$ and a function $v_b \in V_b = \{\phi_b^1, \dots, \phi_b^{N_b}\}$,

$$v_h = \sum_{i=1}^{N_h} \mathbf{v}_h^i \phi_h^i = \sum_{i=1}^{N_{2h}} \mathbf{v}_{2h}^i \phi_{2h}^i + \sum_{i=1}^{N_b} \mathbf{v}_b^i \phi_b^i = v_{2h} + v_b \in V_{2h} + V_b.$$

In this setting it is possible to scale the basis functions ϕ_b^i of the space V_b in such a way that the following two conditions are satisfied:

- There exists a constant $C > 0$ independent of h, r, s such that

$$C^{-1} \leq \|\nabla \phi_b^i\| \leq C, \quad i = 1, \dots, N_b, \quad (3)$$

nVerts	h	$\ u - u_h\ _{L^2}$	rate L_2	$\ \nabla(u - u_h)\ _{L^2}$	rate H_1	angMax
289	h_0	0.00724623	–	0.175665	–	140.334
1089	$h_0/2$	0.00180955	2.0016	0.087845	0.9998	138.116
4225	$h_0/4$	0.000453133	1.9976	0.0439104	1.0004	143.084
16641	$h_0/8$	0.000113451	1.9979	0.0219536	1.0001	152.223
66049	$h_0/16$	0.0000283643	1.9999	0.0109756	1.0002	149.110
263169	$h_0/32$	0.00000709548	1.9991	0.00548762	1.0001	155.643

Table 1: Convergence history of interface problem (1) using mesh adaptation strategy

- There exists a constant $C > 0$ independent of h, r, s , such that for all $v_b \in V_b$

$$|\mathbf{v}_b^i| \leq C \|v_b\|_{\mathcal{N}_i}, \quad i = 1, \dots, N_b, \quad (4)$$

where $\mathcal{N}_I = \{K \in \mathcal{T}_h : x_i \in \overline{K}\}$.

Under these two assumptions it is possible to show the usual bound on the condition number of the system matrix:

Theorem 2. *Assume that (3) and (4) hold. Then there exists a constant $C > 0$ independent of r, s , such that $\text{cond}_2(\mathbf{A}) \leq C h^{-2}$.*

4 Numerical Results

We implemented the method described in Section 2, and tested it for the example where $\Omega = (0, 1)^2$, $\Omega_1 = B(0, 1/2)$, $\Omega_2 = \Omega \setminus \Omega_1$, $\kappa_1 = 1$, $\kappa_2 = 10$, and the right hand side as well as the Dirichlet data were chosen in such a way that the exact solution is known explicitly. The optimal order of convergence stated in Theorem 1 can be observed in Table 4. The interface method was also included in the shape optimization of an electric motor described in [1]. It can be seen from Fig. 2 that smoother and better designs can be achieved by locally modifying the mesh nodes.

5 Conclusion

We presented a local mesh modification strategy which allows to accurately resolve interfaces using the finite element method. We showed a maximal angle condition which ensures optimal order of convergence and presented numerical results for a model problem and for the shape optimization of an electric motor.

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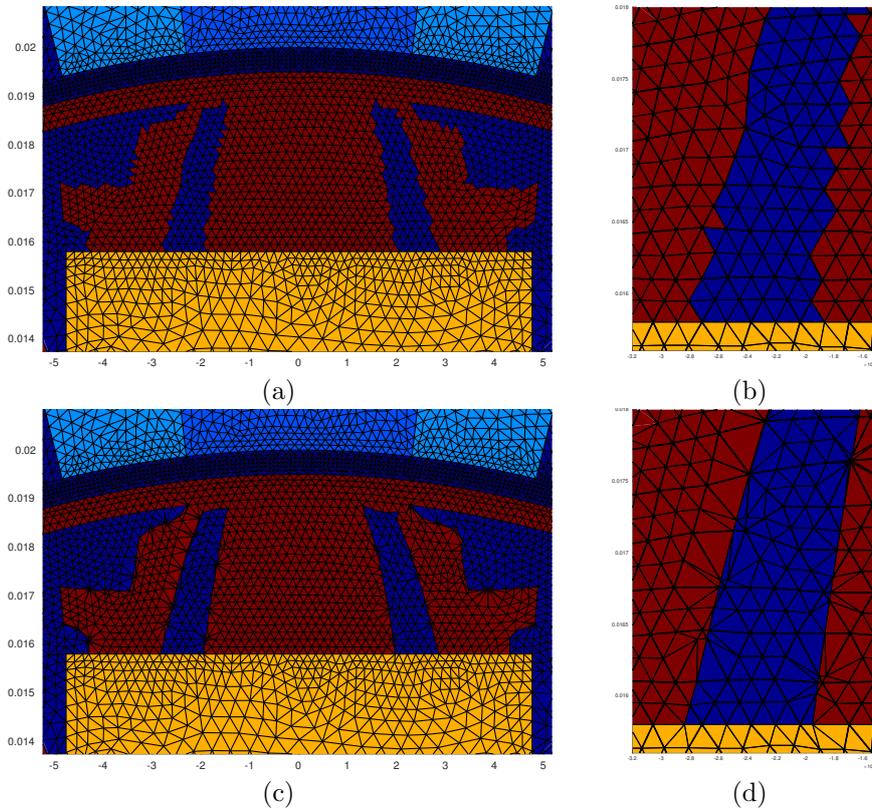


Figure 2: (a) Final design of shape optimization without interface method, objective value $\mathcal{J}(u) \approx 0.0379$. (b) Zoom of (a). (c) Final design of shape optimization with interface method, objective value $\mathcal{J}(u) \approx 0.0373$. (d) Zoom of (c).

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